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Abstract: *The paper presents the basic concepts and methods of data compression. Describes a generalized compression algorithm that uses a system of RGB - model. The necessity of developing a method of compressing data using a dictionary and an algorithm that allows to compress the video stream. This makes it possible to reduce the transmission of data for remote transmission of data (video, remote video conference, etc.).*

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[illegible]

RGB 24 2 24
 , 16 777 215 [4].
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 - «DB_RGB» (Map.txt)
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 4096 RGB , 4096
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 RGB ,
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Каждая таблица базы данных RGB
 является независимой палитрой отличающаяся
 вторым октетом байта

0x0000FF ■ № 4096

...

0x0000F1 ■ № 543

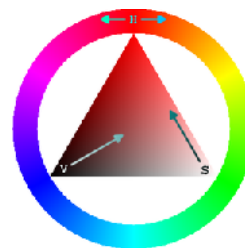
0x0000F0 ■ № 1

для примера дан код красного цвета
 в шестнадцатеричном формате

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RGB



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RGB

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$$\begin{aligned}\dot{r}(t) &= V_r; \\ \dot{V}_r(t) &= V_\theta^2 r^{-1} - (h^2 / p) r^{-2} + U_r; \\ \dot{\theta}(t) &= V_\theta r^{-1}; \\ \dot{V}_\theta(t) &= -V_r V_\theta r^{-1} + U_\theta,\end{aligned}\tag{1}$$

$r = \theta = 0$; $V_r = V_\theta = 0$; $U_r = U_\theta = 0$; $\theta = \gamma + \chi$, $\chi = 0$, $\gamma = 0$, OX .
 $x = r \cos \theta$, $y = r \sin \theta$.
 $h = \frac{2\pi\sqrt{1-e^2} p^2}{(1-e^2)^2 T}$, $\frac{h^2}{p} = GM = \mu = 398600,4 \text{ м}^3/\text{с}^2$, $p =$, $G =$, $M =$, $T =$, $e =$.
 $U_r = U_\theta = 0$.
 (1) ,
 (2) ,
 (3) ,
 (4) .

$$\begin{aligned}r(1 + e \cos \theta) - p &= 0; \\ r^2 \dot{\theta}(t) - h &= r V_\theta - h = 0.\end{aligned}\tag{2}$$

$$\begin{aligned}\psi_1 &= V_r - \varphi_1(r, \theta) = 0, \\ \psi_2 &= V_\theta - \frac{\sqrt{p\mu}}{r} = 0,\end{aligned}\tag{3}$$

$\varphi_1(r, \theta) =$ (« »).
 (3) .

$$\begin{aligned}\dot{r} &= \varphi_1(r, \theta), \\ \dot{\theta} &= \frac{\sqrt{p\mu}}{r^2}.\end{aligned}\tag{4}$$

$$\psi_3 = r - \frac{p}{1 + e \cos \theta} = 0. \quad (5)$$

$$\begin{aligned} & \text{«} \quad \text{»} \quad - \\ & T_3 \dot{\psi}_3(t) + \psi_3 = 0 \quad (4): \end{aligned}$$

$$\varphi_1(r, \theta) = x_3 - \frac{T_3 p e \sin x_2 \sqrt{p\mu} - x_1^3 - 2x_1^3 e \cos x_2 - x_1^3 e^2 \cos x_2^2 + p x_1^2 + p x_1^2 e \cos x_2}{T_3 x_1^2 (1 + 2e \cos x_2 + e^2 \cos x_2^2)}.$$

$$\psi_1 = 0 \quad .$$

$$\begin{aligned} T_1 \dot{\psi}_1(t) + \psi_1 &= 0; \\ T_2 \dot{\psi}_2(t) + \psi_2 &= 0 \end{aligned} \quad (6)$$

$$(1) \quad :$$

$$\begin{aligned} U_r = & (2p x_1^3 e \cos x_2 + p x_1^3 e^2 \cos x_2^2 - 3x_3 T_3 x_1^3 e \cos x_2 - 3x_3 T_3 x_1^3 e^2 \cos x_2^2 - \\ & - 2T_1 x_3 T_3 p e^2 \sin x_2 \sqrt{p\mu} \cos x_2 - 2T_1 x_3 T_3 p e \sin x_2 \sqrt{p\mu} + T_1 e p x_4 x_1^2 \sin x_2 - 3T_1 x_3 x_1^3 e \cos x_2 - \\ & - 3T_1 x_3 x_1^3 e^2 \cos x_2^2 - T_1 x_3 x_1^3 e^3 \cos x_2^2 - 3T_1 x_4^2 T_3 x_1^2 e \cos x_2 - 3T_1 x_4^2 T_3 x_1^2 e^2 \cos x_2^2 - \\ & - T_1 x_4^2 T_3 x_1^2 e^3 \cos x_2^3 + 2T_1 e^2 p x_4 T_3 \sin x_2^2 \sqrt{p\mu} + T_1 e p x_4 T_3 \cos x_2 \sqrt{p\mu} + T_1 e^2 p x_4 T_3 \cos x_2^2 \sqrt{p\mu} + \\ & + T_1 e^2 p x_4 x_1^2 \sin x_2 \cos x_2 + 3T_1 \mu T_3 x_1 e \cos x_2 + 3T_1 \mu T_3 x_1 e^2 \cos x_2^2 + T_1 \mu T_3 x_1 e^3 \cos x_2^3 + \\ & + x_1 T_3 p e \sin x_2 \sqrt{p\mu} + x_1 T_3 p e^2 \sin x_2 \sqrt{p\mu} \cos x_2 - x_3 T_3 x_1^3 e^3 \cos x_2^3 - 3x_1^4 e^2 \cos x_2^2 - \\ & - 3x_1^4 e \cos x_2 - T_1 x_3 x_1^3 - x_1^4 - x_3 T_3 x_1^3 - T_1 x_4^2 T_3 x_1^2 + p x_1^3 + T_1 \mu T_3 x_1 - \\ & - x_1^4 e^3 \cos x_2^3) / (T_1 T_3 x_1^3 (3e \cos x_2 + 1 + 3e^2 \cos x_2^2 + e^3 \cos x_2^3)); \\ U_\theta = & - \frac{T_2 \sqrt{p\mu} x_3 - T_2 x_3 x_4 x_1 + x_4 x_1^2 - \sqrt{p\mu} x_1}{T_2 x_1^2}. \end{aligned} \quad (7)$$

:

$$0,5(V_r^2 + V_\theta^2) - \frac{\mu}{r} - \frac{(e^2 - 1)\mu}{2p} = 0. \quad (8)$$

:

$$\begin{aligned} \psi_1 &= V_r - \varphi_1(r, \theta) = 0, \\ \psi_2 &= 0,5(V_r^2 + V_\theta^2) - \frac{\mu}{r} - \frac{(e^2 - 1)\mu}{2p} = 0. \end{aligned} \quad (9)$$

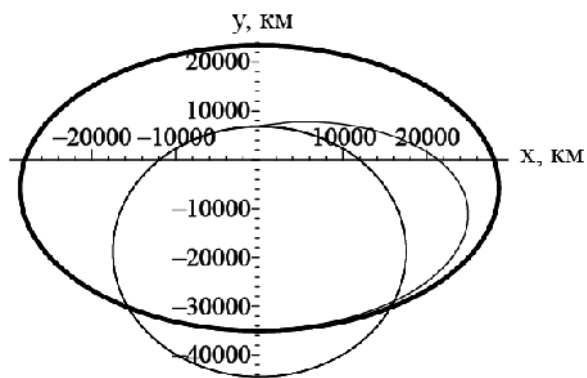
$$\varphi_1(r, \theta) - \quad (\text{«} \quad \text{»} \quad).$$

$p=6297,8$
(7).

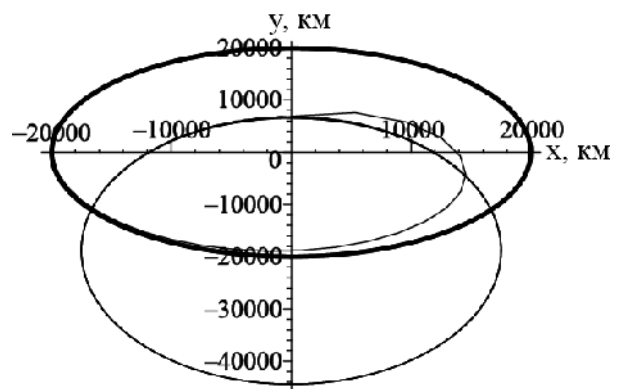
. 2
 $e=0,87; p=6297,8$

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 $e=0,87;$
 $e=0,2; p=28000$

: $e=0; p=20000$



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[4].

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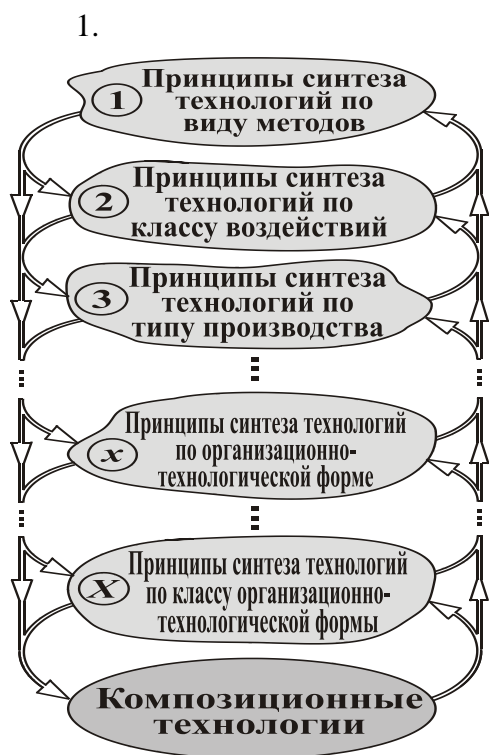
4.).

5. ? - (

KOT –
T_x – *x* –
X –

$$Str_k\{T, A\} = \bigotimes_{x=1}^X Str_x\{T_x, A_x\}, \quad (2)$$

$Str_k\{T, A\}$ - , T
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 $Str_x\{T_x, A_x\}$ - x - , T_x
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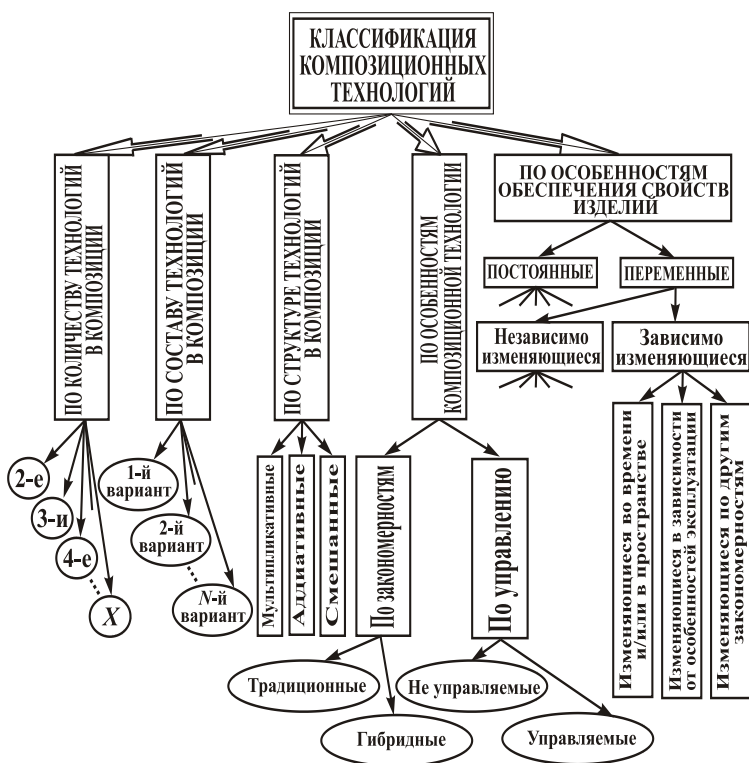


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 : , 2001. – 368 . ISBN 5-217-03061-5. 5. . . -
 - : , 2009. – 346
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 », 2011. . 23 - 31. ISSN 0321-0499.

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$$\begin{aligned}
& \dot{\theta}(t) = \omega; \\
& J\dot{\omega}(t) = M_{ynp} - k_{mp}\omega,
\end{aligned} \tag{1}$$

where θ — angular displacement, ω — angular velocity, M_{ynp} — nominal torque, k_{mp} — damping coefficient, J — moment of inertia.

The system is controlled by a feedback law:

$$\omega_0 = A_{max} \sin(\theta), \tag{2}$$

where A_{max} is the maximum amplitude of the control signal.

The solution of the system (1) can be found analytically:

$$\begin{aligned}
\dot{y}_1(t) &= (A_{max}^2 - y_1^2 - y_2^2)y_1 + \omega_0 y_2, \\
\dot{y}_2(t) &= (A_{max}^2 - y_1^2 - y_2^2)y_2 - \omega_0 y_1.
\end{aligned} \tag{2}$$

(2) , (1) (2).
« »

$$\psi_1 = \omega - b(\theta - y_1) = 0. \quad (3)$$

$$\dot{\theta}(t) = b(\theta - y_1);$$

$$\dot{y}_1(t) = (A_{\max}^2 - y_1^2 - y_2^2)y_1 + \omega_0 y_2,$$

$$\dot{y}_2(t) = (A_{\max}^2 - y_1^2 - y_2^2)y_2 - \omega_0 y_1.$$

$b < 0$ θ A_{\max} - ω_0 .

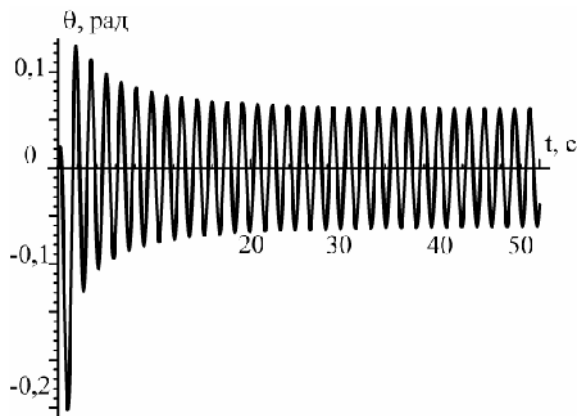
$$T_1 \dot{\psi}_1(t) + \psi_1 = 0$$

(1) (2), :

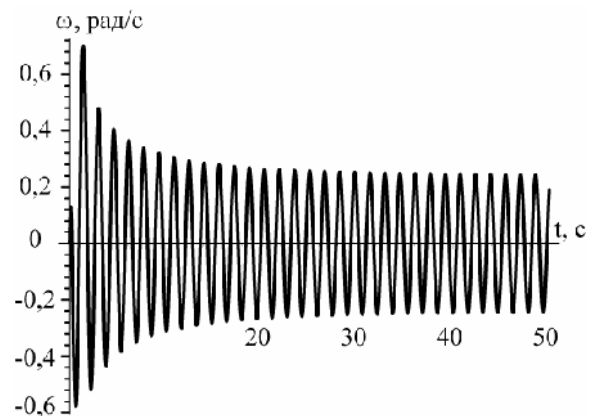
$$M_{\text{ynp}} = k_{\text{mp}} \omega + Jb(\omega - (A_{\max}^2 - y_1^2 - y_2^2)y_1 - \omega_0 y_2) - \frac{J}{T_1}(\omega - b(\theta - y_1)). \quad (4)$$

(1), (2), (4), -

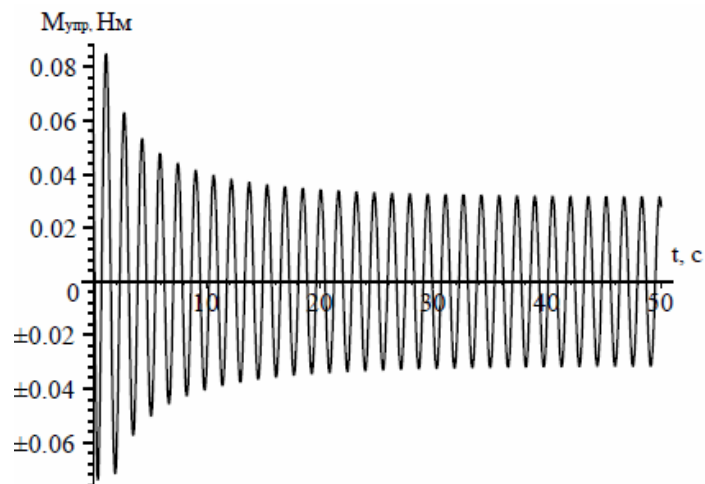
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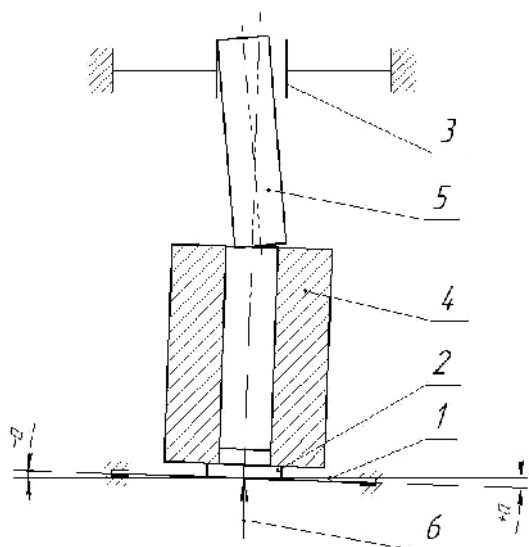
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 , 1997. 4. . —
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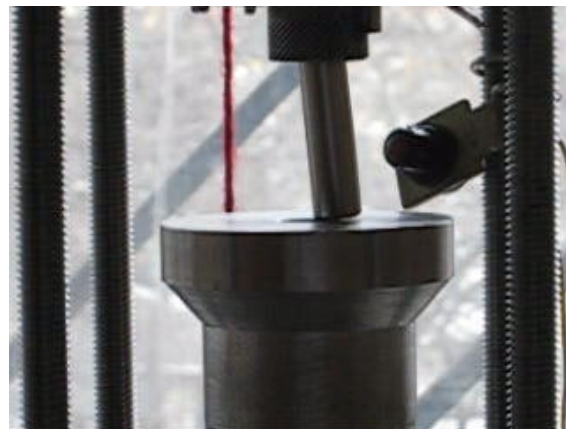
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 ./ : +38(062) 3010804; E-mail: olga_kulbida@mail.ru

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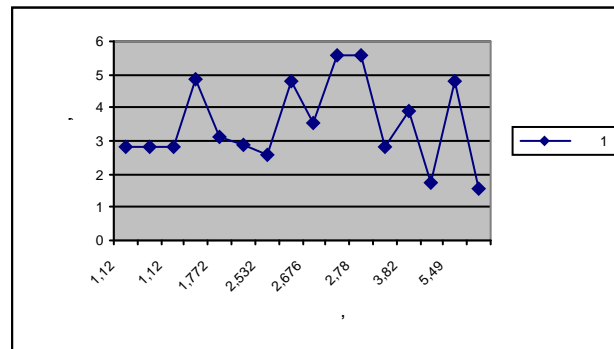
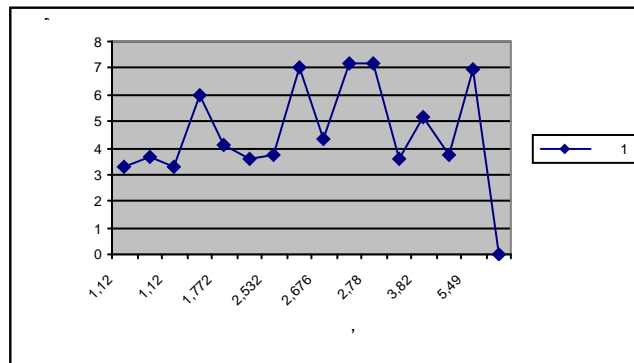
Ø10 9/f9, Ø10 9/d9, Ø10 9/e9;
: 1,6 45°, 1,6 30°;
: 1,6 45°, -
R1,6, 1,6 30°, -

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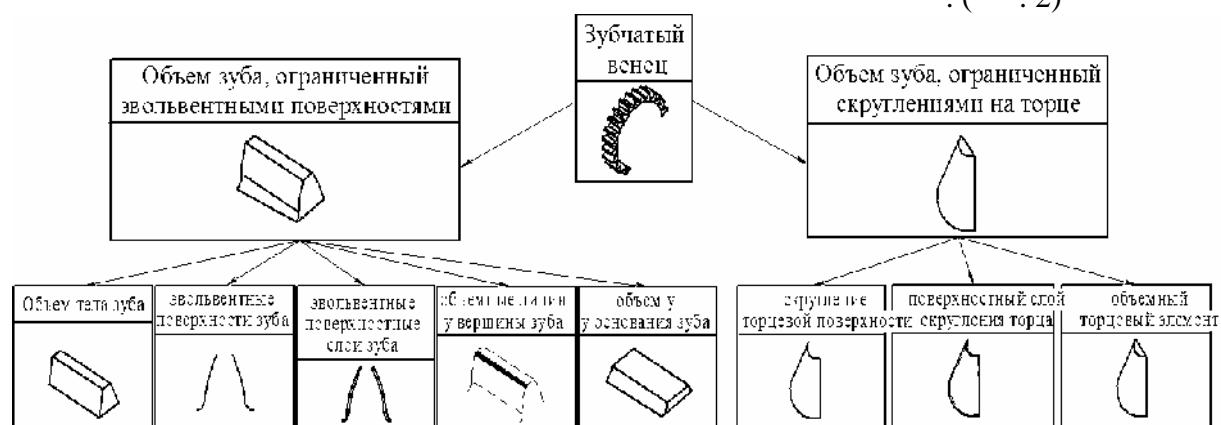
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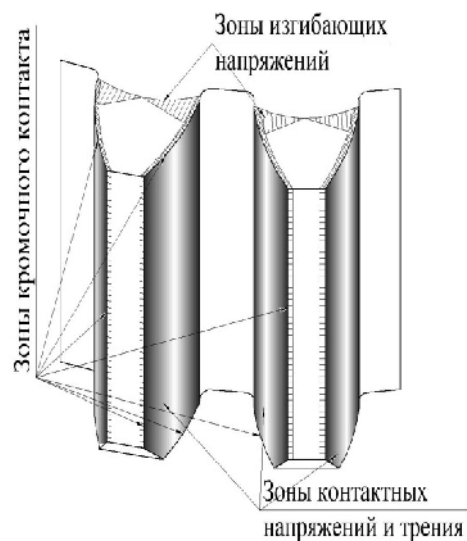
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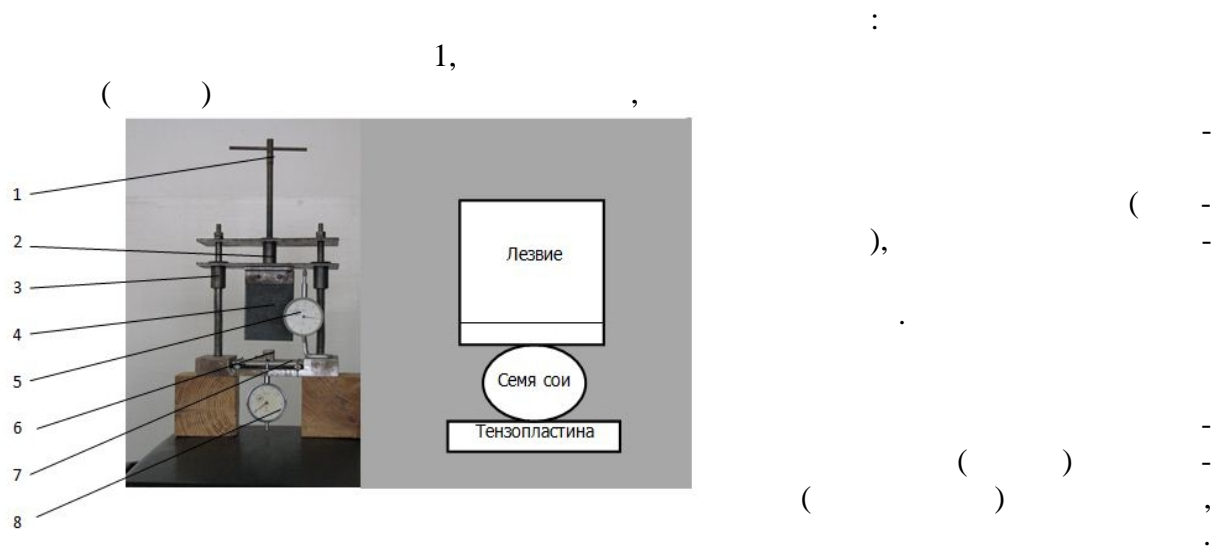
: , 2009. . 38.

127-132. **3.** . -.: « », 1957. – 263 .

631.363.21

1. $W = 12,5\%$ (W + 30° , 1, 0,25 – 0,30),

1,5 %)



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1 – ; 2 – ; 3 – ; 4 – (); 5 – () ; 6 – ; 7 – ; 8 – .

0,01 ; (-10; 0-10); 8 () ; 2 – 3 ; (-10) ,

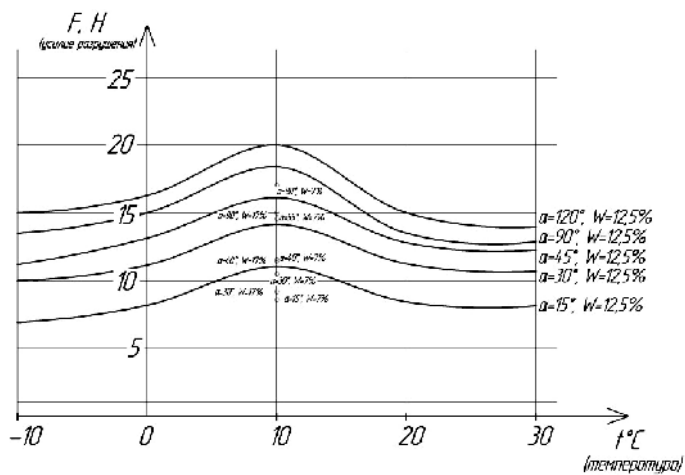
$$W = 12,5\% \quad t^{\circ} \quad F(t^{\circ})/W = \text{const}, \quad -10^{\circ} \quad +30^{\circ} ,$$

1.

t, °C					
	15°	30°	45°	90°	120°
-10	6,5	9,0	11,5	13,0	14,0
0	8,0	10,0	13,0	15,0	16,0
10	10,0	14	15,5	17,0	20,0
20	9,0	10,0	12,5	13,0	14,0

30	8	9,5	11	12,5	13,8
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$$v = \frac{V}{V_0}, \quad (1)$$

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$$V -$$

$$V_0 -$$

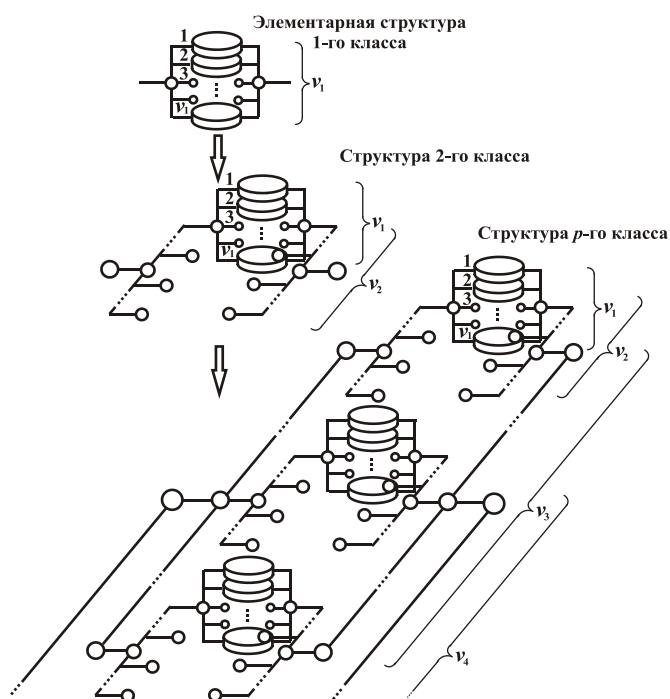
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 (1)

$$v$$

$$V = V_0 .$$

$$\left. \begin{array}{l} v \rightarrow \max , \\ V_0 \rightarrow \min . \end{array} \right\}$$

$$P = 1/v ;$$



. 1.

y a ,

$$Str = \{y, a\},$$

Str -

y -
 a -

y a

$$\begin{aligned} y &= \{y_1, y_2, \dots, y_v\}, \\ a &= \{a_1, a_2, \dots, a_{v_i}\} \end{aligned}$$

y_η - η -

y ;

a_η - η - y .

$y = a$.

(1)

1-

2- , - 3-

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$$v = \prod_{k=1}^p v_k,$$

v_{ik} - k - ;

(1),

$$\frac{\prod_{k=1}^p v_k}{T}.$$

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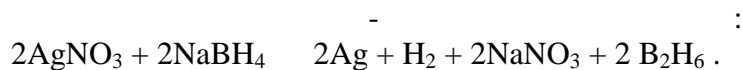
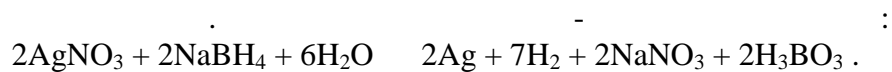
$$Str = \{Str_1, Str_2, ..., Str_p\},$$

Str_k - k -

$$Str = \bigcup^{v_P} \dots \bigcup^{v_2} \bigcup^{v_1} y_\eta.$$

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:1. . . , - : - , 2002. – 328 . ISBN 5-217-03119-0. 2. . . - : - , 2006. – 256 . ISBN 5-217-03340-1. 3. . . , 1. . . : « », 2007. – 518 . ISBN 966-87-2. 4. . . , . . . , . . . : « », 2005. – 559 . ISBN 966-7108-91-0. 5. . . - : - : - , 2001. – 622 . ISBN 5-89594-066-8.



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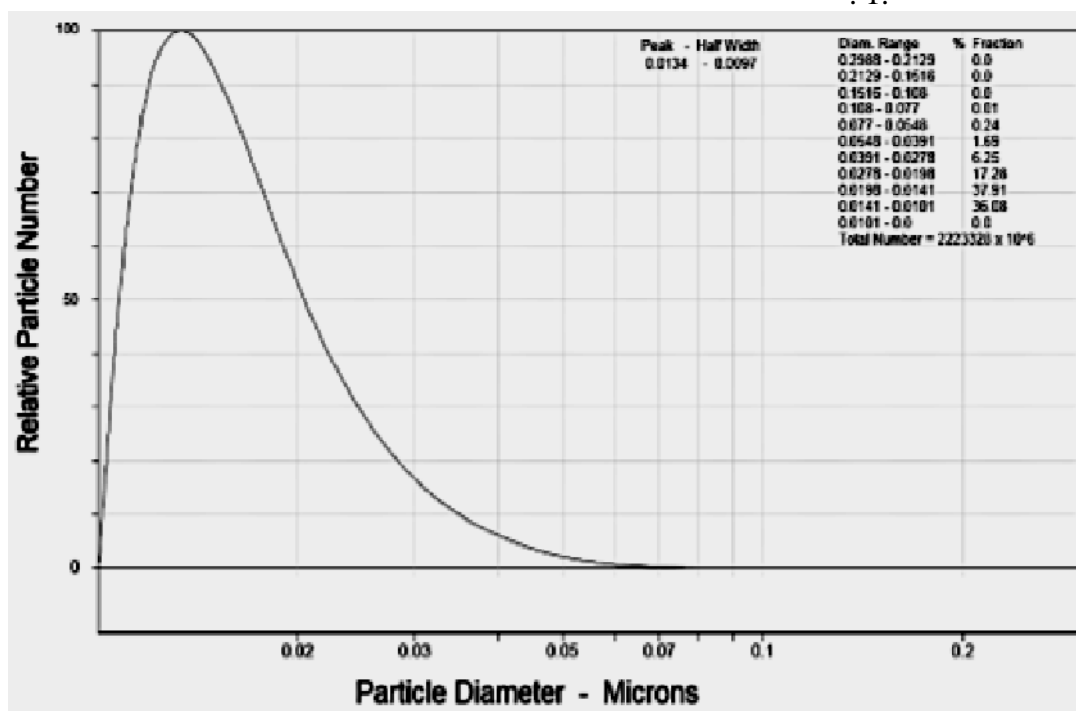
[7],

2.1.

$$C_M = 0,001 \quad /$$

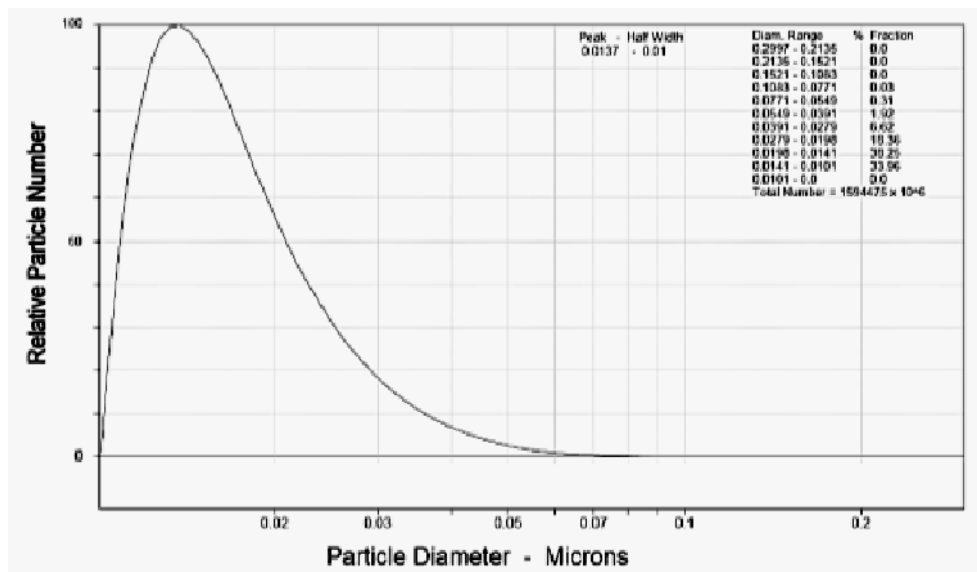
$$[\text{AgNO}_3]/[\text{NaBH}_4] = 1/20,$$

3646/51.
0⁰ .
0,1 %.
. 1.



.1.
[AgNO₃]/[NaBH₄] = 1/20 (C_M = 0,001 /)
3646/51 0,1 %

(. 2).



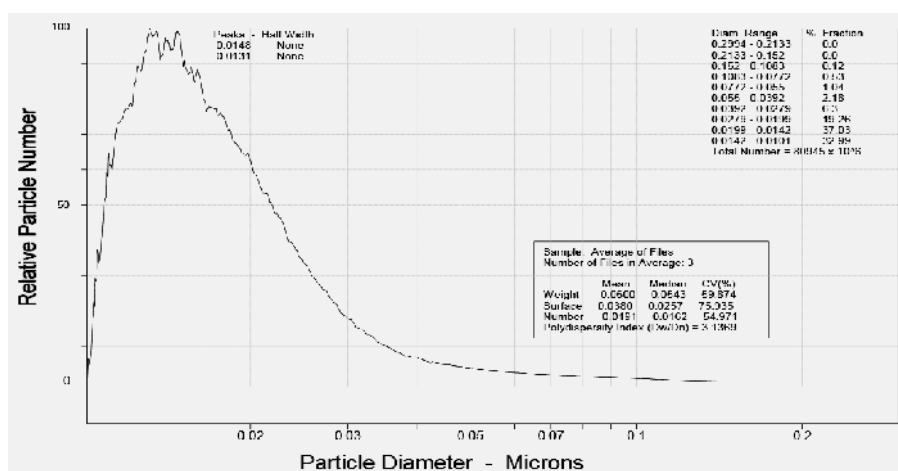
.2.

$[AgNO_3]/[NaBH_4] = 1/20$ ($C_M = 0,001$ /)

3646/51 0,1 %

2.2.

($C_M = 0,0003$ /).

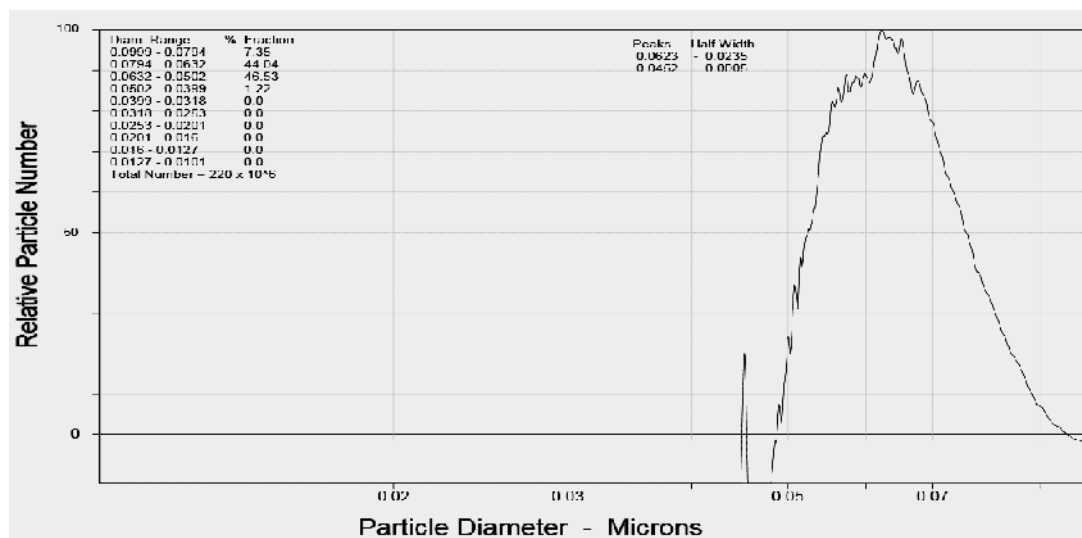


.3.

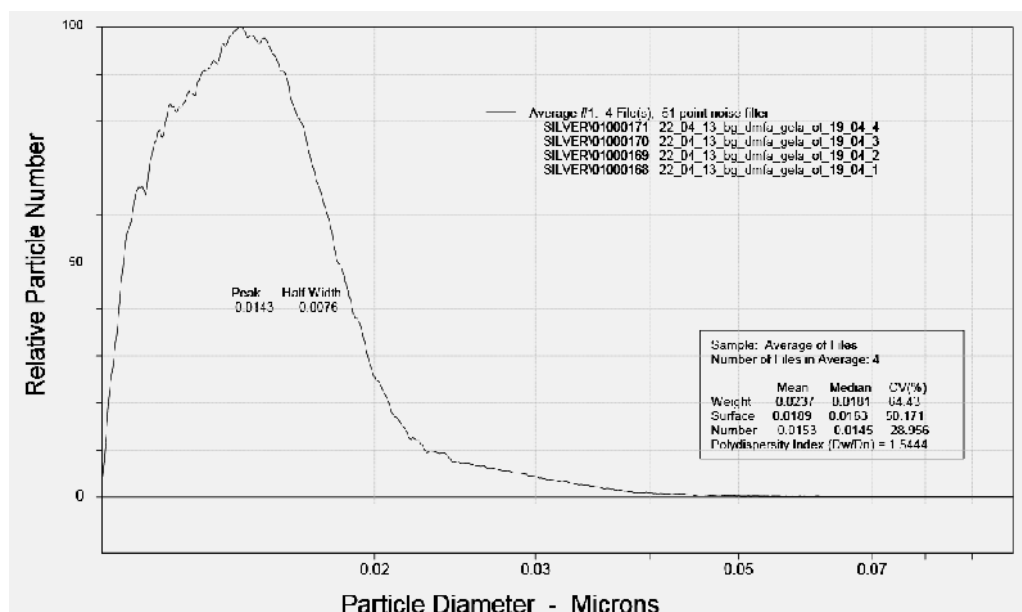
$[AgNO_3]/[NaBH_4] = 1/20$ ($C_M = 0,001$ /)

$C_M = 0,0003$ /

$C_M = 0,001$ / , $[AgNO_3]/[NaBH_4] = 1/20$,
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.4.
 $[AgNO_3]/[NaBH_4] = 1/20$ ($C_M = 0,001$ /)
 $C_M = 0,0003$ /
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 ($C_M = 0,0001$ /)
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 $[AgNO_3]/[NaBH_4] = 1/4$
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.5.
 $[AgNO_3]/[NaBH_4] = 1/4$ ($C_M = 0,0001$ /) N, N-
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 Marzan L.M. Reduction of silver nanoparticles in DMF. Formation of monolayers and stable
 colloids // Pure Appl. Chem. 2000. Vol. 72. P. 83-90.

621.75

COLD PLASTIC DEFORMATION OF THE BEARING RINGS RACEWAY**Pruteanu O.¹, Carausu C.¹, Gramescu T.¹, Nedelcu D.¹***(1 – UT, Iasi, Romania, Romania)*

Abstract: This paper presents comparative results from cold plastic deformation on outer ring raceway of the two types of bearings: 6211 and 6309. The experimentation conditions, the variation diagrams of the shape deviation (circularity) and of the roughness of the raceway of the outer rings depending on deformation force, deformation feed and rotation speed of the roller. After the mathematic modeling of the experimental results, the comparative analysis and the conditions concerning the optimum processing conditions are presented.

Key words: plastic deformation, circularity, roughness, mathematic modeling.

1. Introduction

The cold plastic deformation is characterized by high precision in terms of dimension and shape, superior quality of the processed surfaces, roughness and superficial layer, metal economy, increased productivity, etc.

In literature there are works [2, 3] which present the influence of some technological factors on the quality parameters of the processed surface. Paper [1] presents calculation relations established by various researchers depending on the considered criterion: geometry of the tool and of the piece [Maier, Schneider], the roughness of the run surface [Herold], the work parameters [Bokoiavlenski] etc.

2. Experimental conditions

Experimental research was realized at SC Rulmen i SA Bârlad in the following conditions:

- material: 100Cr6;
- unfinished goods: warm forged and laminated;
- outer rings for the types of bearings 6211 and 6309;
- machine: CRF-120 OR, with the work parameters presented in table 1;
- tools: specific to the machine and to the type of bearing – mandrel and deformation roller;
- measurement devices: Taylor Hobson Formtalysurf.

Table 1. Work parameters of the machine CRF-120 OR

Parameters	Range	Experimental values
Deformation force, F [daN]	12100 - 17600	12100; 13200; 14300; 15400; 16500; 17600
Feed rate, s [mm/min]	30 – 50	30; 35; 40; 45; 50
Rotation speed of the roller, n [rpm]	85 – 125	85; 90; 95; 100; 110; 120

3. Experimental findings

Experimental research was conducted on the two types of outer rings from bearings 6211 and 6309 by variation of one parameter at the values mentioned in table 1 and the maintaining of the other two parameters constant. The results are presented in table 2.

Table 2. Experimental results

Independent parameter	6211-10		6309-10	
	Circularity μm	Roughness μm	Circularity μm	Roughness μm
Force F [daN]	Variation of the deformation force, F [daN] / constant : n = [90 rpm] : s = 40 [mm/min]			
12100	130	0.28	150	0.24
13200	310	0.35	260	0.32
14300	180	0.33	240	0.34
15400	275	0.29	310	0.27
16500	300	0.28	290	0.30
17600	160	0.30	180	0.32
Feed rate s [mm/min]	Variation of the feed rate, s [mm/min] constant P= 15400[daN], n= 90 [rpm]			
30	255	0.085	170	0.10
35	195	0.10	185	0.12
40	180	0.15	210	0.22
45	210	0.12	205	0.14
50	220	0.18	200	0.19

The figures 1-2 present the variation of the quality parameters (roughness and circularity) depending on the work parameters (deformation force and feed rate).

4. Mathematical modelling of the experiment results

For the deduction of the relations of work parameters influences on quality parameters, the Table Curve 2D v5.01 is used.

The correlation coefficient R^2 was deducted, the estimated error, the significance test F was applied, and through comparison the most appropriate model was established for the experiment data [4].

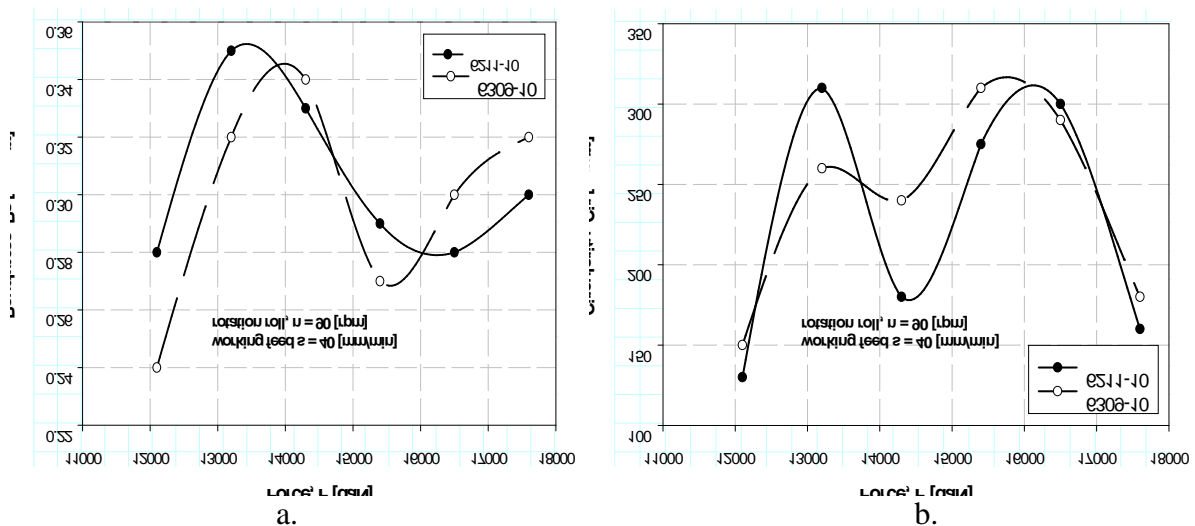


Fig. 1. Variation of the roughness (a) and circularity (b) with the deformation force, F

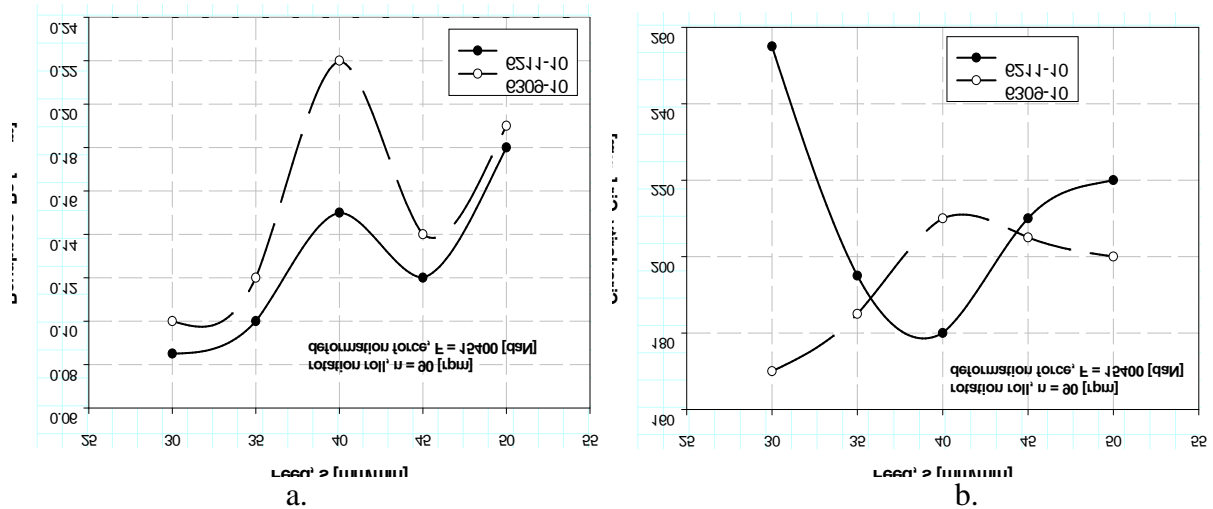


Fig. 2. Variation of the roughness (a) and circularity (b) with the feed, s

The mathematic expression of the models which best describe the factor influences – deformation force and feed rate - on roughness and circularity are presented in table 3.

The graphical representation of these functions and the experimental values are presented in figures 3-4.

Table 3. The mathematic expression of the models which describe the factor influences on roughness and circularity

Parameter	Factor	Ring	Mathematical expression	R ²	Significance test F
Roughness, Ra	Force, F	6211-10	$Ra = a + \frac{b}{F} + c \cdot \frac{\ln F}{F^2} + \frac{d}{F^2}$, with: a = 22.656; b = -1336876.8; c = 9.8856e+9; d = -8.003e+10	0.997	239.69
		6309-10	$Ra = a + b \cdot \ln F + c \cdot \frac{\ln F}{F} + \frac{d}{F}$, with: a = -1187.08; b = 102.29; c = 1516398.7; d = -11524812	0.832	3.3
	Feed, s	6211-10	$Ra = a + b \cdot e^s + \frac{c}{s^2}$, with: a = 0.1752; b = 7.1357e-24; c = -80.72	0.835	5.078
		6309-10	$Ra = a + \frac{b}{\ln s} + c \cdot \frac{\ln s}{s}$, with: a = -24.908; b = 148.51; c = -164.58	0.514	1.058
Circularity, Cir	Force, F	6211-10	$Cir = a + b \cdot F^2 + c \cdot F^4$, with a = -623.85; b = 7.777e-6; c = 1.674e-14	0.835	5.079
		6309-10	$Cir = a + b \cdot F^2 + c \cdot F^4$, with a = -739.46; b = 8.888e-6; c = 1.901e-14	0.514	1.06
	Feed, s	6211-10	$Cir = a + b \cdot s^2 + c \cdot s^4$, with a = 582.204; b = -1177014; c = 3708704.6	0.835	5.079
		6309-10	$Cir = \frac{1}{a + \frac{b}{\ln s} + c \cdot \frac{\ln s}{s}}$, with: a = 0.4744; b = -2.77; c = 3.055	0.514	1.058

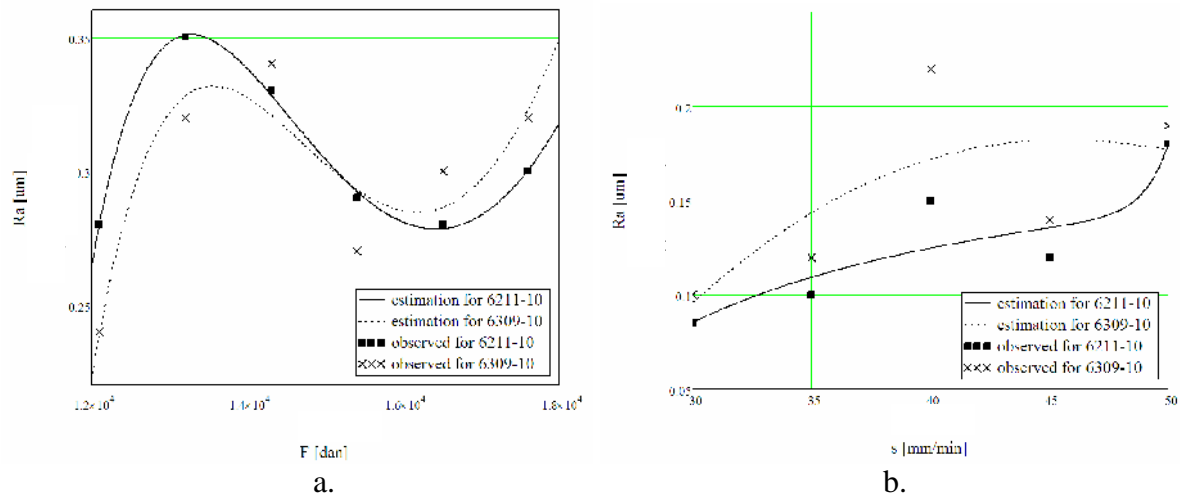


Fig. 3. Estimated influences of deformation force (a) and feed rates (b) on roughness, Ra

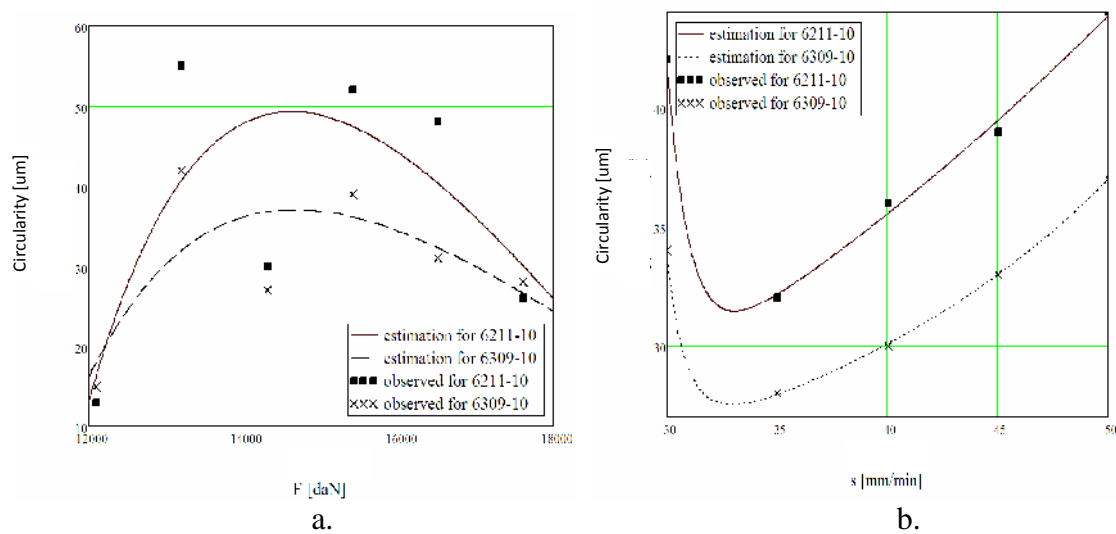


Fig. 4. Estimated influences of deformation force (a.), feed rates (b.) on circularity

5. Conclusions

Following the graphic representation of the values of the quality parameters depending on the work parameters of the machine to be processed by cold plastic deformation, one can reach the following conclusions:

- the increase of the deformation force, there occurs an increase on the roughness for both types of bearings, fig. 1-a, with smaller values being registered at the 6309-10 type as a result of a bigger contact surface between the race and the roller;
- the shape, circularity deviations, fig. 1-b, generally increase by the increase of the deformation force up to the force of 16500 [daN] with one exception at the force of 14300 [daN], after which one may notice a relatively small increase of deviations. The cause of this variation can be explained by the vibrations and self-vibrations in the technological system;
- feed rate, the roughness of the processed surfaces also increases, fig. 2-a, with a bigger influence on the ring 6309-10, as well as on shape deviations, fig. 2-b, with a reduction around the value of 35-40 mm/min. This tendency can be explained via the bigger sizes of the contact surface between the unfinished goods and the roller.

[2],

$$h = 2 \frac{K}{HB^\gamma} \int_0^{\varphi_k} P_{ai}^\varphi(\varphi_i) L_i(\varphi_i) d\varphi_i,$$

[3].

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30%.

Ra 3,2...6,3 .

30,

(890')

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150° ,

150° ,

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