

## DYNAMICS OF A POINT OF VARIABLE WEIGHT AND THE DIFFERENTIAL EQUATIONS OF MOVEMENT OF A LIQUID IN PNEUMATIC HYDRAULIC PATHS PUMP HOUSE-AIR-LIFT INSTALLATIONS DURING START-UP.

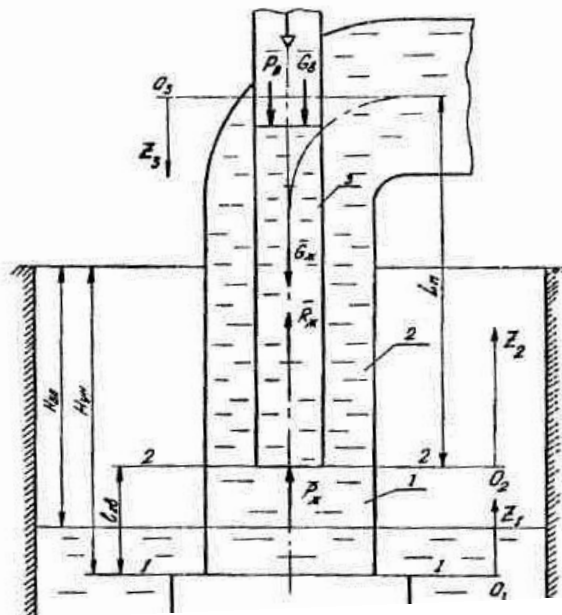
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Mesherskiy has established that if the weight of a point changes during movement the basic differential equation of movement of Newton is replaced with the following equation of movement of a point of variable weight:

$$m \frac{d\bar{v}}{dt} = \bar{F} + \bar{R}, \quad \text{Where } \bar{F} \text{ and } \bar{R} = \frac{dm}{dt} \bar{U}_r \text{ — the set and jet forces.}$$

Let's consider transients in pneumatic hydraulic paths pump house-air-lift installations which circuit is resulted in figure where the 1- delivery pipeline of the pump and the having pipeline air-lift ; 2- elevating pipe; a 3- air pipe.



The settlement circuit pump -air-lift adjustment

Transients during start-up are considered in the assumption, that the pump already works and submission of compressed air in an air pipe

air-lift starts to be carried out. The period of replacement of a liquid from an air pipe by the compressed air down to his break through the amalgamator in an elevating pipe air-lift is investigated. For drawing up of the differential equation of movement of a liquid in an air pipe we use the equation of dynamics of a body of the variable weight, written down in projections to an axis  $Z_3$ :

$$m\ddot{Z}_3 = \sum F_{kz3}^e + \frac{dm}{dt} \cdot U_{z3},$$

Where  $m$ -weight of a liquid in an air pipe, kg;  $Z_3$  - coordinate of a free surface of a liquid in an air pipe;  $\sum F_{kz3}^e$  - the sum of projections to an axis  $Z_3$  of the external forces working on a liquid moving in an air pipe, H;  $U_{z3}$  - a projection to an axis  $Z_3$  of a vector of speed of weight of water moving in an air pipe during its branch, m/s.

$$\sum F_{kz3}^e = P_{\delta 3} + G_{\delta} + G_{\lambda c} - P_{\lambda c} - R_{\lambda c},$$

Where  $P_{\delta 3}$  - force of pressure of compressed air, H;  $G_{\delta}$  - a gravity of volume of air, H;  $G_{\lambda c}$  - a gravity of volume of a liquid, H;  $P_{\lambda c}$  - force, pressure working on weight of a liquid in an air pipe on the part of the bringing pipeline;  $R_{\lambda c}$  - force of resistance to movement of a liquid in an air pipe, H.

$$\ddot{Z}_3 = \frac{1}{N_1 + N_2 Z_3} \cdot \left( \frac{N_3}{\dot{Z}_3} + N_4 \frac{Z_3}{\dot{Z}_3} + N_6 Z_3^2 + N_8 + N_9 P_2 \right) + N_5 + N_7 \dot{Z}_3^2,$$

$$\text{Where } N_1 = \rho \cdot L_n F_{\delta 3}, N_2 = -\rho \cdot F_{\delta 3},$$

$$N_3 = V_0 P_a, N_4 = \rho_0 V_0 g,$$

$$N_5 = g, N_6 = -\rho \cdot F_{\delta 3}, N_7 = -\frac{\lambda_3}{(2d_{\delta 3})}, N_8 = -P_a F_{\delta 3}, N_9 = -F_{\delta 3}.$$

Thus, in view of the equation of indissolubility of a stream of movement of a liquid in pneumatic hydraulic paths pump house-air-lift to installation it is described by the following system of the nonlinear differential equations of the second order:

$$\begin{cases} D_1\ddot{Z}_1 + D\dot{Z}_1^2 + D_3\dot{Z}_1 + D_4Z_1 = P_2 + D_5, \\ M_1\ddot{Z}_2 + M_2\dot{Z}_2^2 = P_2 + M_3 \\ \ddot{Z}_3 = \frac{1}{(N_1 + N_2Z_3)} \cdot \left( \frac{N_3}{\dot{Z}_3} + N_4 \frac{Z_3}{\dot{Z}_3} + N_6\dot{Z}_3^2 + N_8 + N_9P_2 \right) + N_5 + N_7\dot{Z}_3^2, \\ \dot{Z}_1 F_{x6} + \dot{Z}_3 F_{63} = \dot{Z}_2 F_n \end{cases}$$

Where  $P_2$ - hydrostatic pressure in section 2-2,  $F_{x6}$ - the area of section of the bringing pipeline,  $m^2$ ,  $V_0$ -productivity of the compressor at atmospheric pressure  $P_a=9,8 \cdot 10^4 \text{ Pa}$ ,  $m^3/c$ ,  $\rho_0$ - density of air under normal conditions,  $\text{кг}/m^3$ .  $\lambda_3$ -coefficient of hydraulic resistance at movement of a liquid in an air pipe;  $d_{63}$ -diameter of an air pipe, m.

