## DYNAMICS OF A POINT OF VARIABLE WEIGHT AND THE DIFFERENTIAL EQUATIONS OF MOVEMENT OF A LIQUID IN PNEUMATIC HYDRAULIC PATHS PUMP HOUSE-AIR-LIFT INSTALLATIONS DURING START-UP

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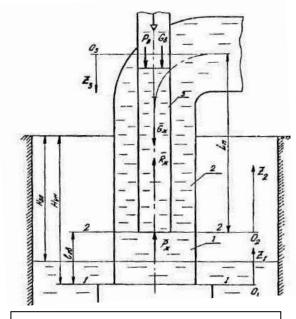
Mesherskiy has established that if the weight of a point changes during movement the basic differential equation of movement of Newton is replaced with the following equation of movement of a point of variable weight:

$$m\frac{d\overline{v}}{dt} = \overline{F} + \overline{R}$$

Where  $\overline{F}$  and  $\overline{R} = \frac{dm}{dt} \overline{U_r}$  - the set and jet forces.

Let's consider transients in pneumatic hydraulic paths pump house-air-lift installations which circuit is resulted in figure where the 1-delivery pipeline of the pump and the having pipeline air-lift; 2-elevating pipe; a 3-

air pipe.



The settlement circuit pump - air-lift adjustment

Transients during start-up are considered in the assumption, that the pump already works and submission of compressed air in an air pipe air-lift starts to be carried out. The period of replacement of a liquid from an air pipe by the compressed air down to his break through the amalgamator in air-lift elevating pipe an investigated. For drawing up of the differential equation of movement of a liquid in an air pipe we use the equation of dynamics of a body of the variable weight, written down in projections to an axis  $Z_3$ :

$$m\ddot{Z}_3 = \sum F_{kz_3}^e + \frac{dm}{dt} \cdot U_{z_3},$$

Where m-weight of a liquid in an air pipe,  $kg; Z_3$  - coordinate of a free surface of a liquid in an air pipe;  $\sum F_{kz_3}^e$  - the sum of projections to an axis  $Z_3$  of the external forces working on a liquid moving in an air pipe, H;  $U_{z_3}$  - a projection to an axis  $Z_3$  of a vector of speed of weight of water moving in an air pipe during its branch, m/s.

$$\sum F_{kz_3}^e = P_{e3} + G_{e} + G_{HC} - P_{HC} - R_{HC},$$

Where  $P_{e3}$  - force of pressure of compressed air, H;  $G_{e}$  - a gravity of volume of air, H;  $G_{c}$  - a gravity of volume of a liquid, H;  $P_{c}$  - force, pressure working on weight of a liquid in an air pipe on the part of the bringing pipeline;  $R_{c}$  - force of resistance to movement of a liquid in an air pipe, H.

$$\ddot{Z}_3 = \frac{1}{N_1 + N_2 Z_3} \cdot (\frac{N_3}{\dot{Z}_3} + N_4 \frac{Z_3}{\dot{Z}_3} + N_6 Z_3^2 + N_8 + N_9 P_2) + N_5 + N_7 \dot{Z}_3^2,$$

Where  $N_1 = \rho \cdot L_n F_{63}$ ,  $N_2 = -\rho \cdot F_{63}$ ,

$$N_3 = V_0 P_a, N_4 = \rho_0 V_0 g,$$

$$N_5 = g, N_6 = -\rho \cdot F_{e3}, N_7 = -\frac{\lambda_3}{(2d_{e3})}, N_8 = -P_a F_{e3}, N_9 = -F_{e3}.$$

Thus, in view of the equation of indissolubility of a stream of movement of a liquid in pneumatic hydraulic paths pump house-air-lift to installation it is described by the following system of the nonlinear differential equations of the second order:

$$\begin{cases} D_1\ddot{Z}_1 + D\dot{Z}_1^2 + D_3\dot{Z}_1 + D_4Z_1 = P_2 + D_5, \\ M_1\ddot{Z}_2 + M_2\dot{Z}_2^2 = P_2 + M_3 \\ \ddot{Z}_3 = \frac{1}{(N_1 + N_2Z_3)} \cdot (\frac{N_3}{\dot{Z}_3} + N_4\frac{Z_3}{\dot{Z}_3} + N_6\dot{Z}_3^2 + N_8 + N_9P_2) + N_5 + N_7\dot{Z}_3^2, \\ \dot{Z}_1F_{x6} + \dot{Z}_3F_{63} = \dot{Z}_2F_n \end{cases}$$

Where  $P_2$ - hydrostatic pressure in section 2-2,  $F_{x\theta}$ - the area of section of the bringing pipeline,  $M^2, V_0$ -productivity of the compressor at atmospheric pressure  $P_a = 9.8 \cdot 10^4 \, \mathrm{Pa}$ ,  $M^3/\mathrm{c}$ ,  $\rho_0$ - density of air under normal conditions,  $\mathrm{KF}/M^3$ .  $\lambda_3$ -coefficient of hydraulic resistance at movement of a liquid in an air pipe;  $d_{\theta 3}$ - diameter of an air pipe, m.